

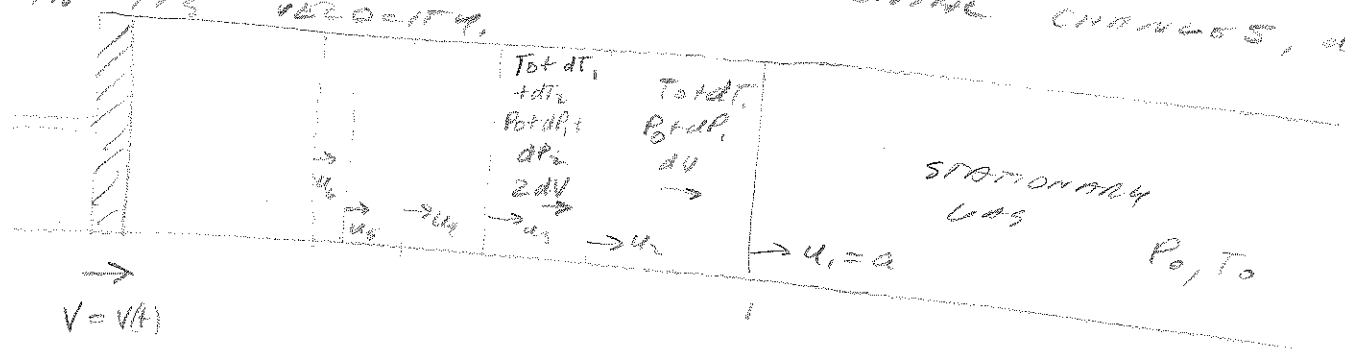
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D

LECTURE III
NORMAL SHOCKS; OBLIQUE SHOCKS

(I) NORMAL SHOCKS

A) SHOCK FORMATION

WE CAN GET AN INTUITIVE UNDERSTANDING OF HOW NORMAL SHOCKS FORM BY CONSIDERING THE ACCELERATION OF A PISTON INTO AN AIR-FILLED TUBE. TO VISUALIZE THE PROCESS, IMAGINE THAT THE PISTON ACCELERATES FROM 0 TO V THROUGH A SERIES OF INCREMENTAL CHANGES, dV , IN ITS VELOCITY.



WAVE ① FORMED WHEN PISTON ACCELERATES FROM 0 TO dV ; ITS SPEED IS $a = \sqrt{\gamma R T_0}$, WHERE T_0 IS THE INITIAL GAS TEMP. THE VELOCITY OF THE GAS BEHIND WAVE ① AND IN FRONT OF WAVE ② IS dV . IN ADDITION, DUE TO A SMALL COMPRESS, THE PRESSURE IS $P_0 + dP_0$. WAVE ② FORMS WHEN THE PISTON VELOCITY JUMPS FROM dV TO $2dV$. THE SPEED OF WAVE ② RELATIVE TO THE MOVING GAS BEHIND WAVE ① IS $a + da$, WHERE da IS > 0 SINCE $T = T_0 + dT_1$.

2/3/00②

THUS THE ABSOLUTE SPEED OF WAVE ②
(RELATIVE TO THE FIXED OUTER WALLS) IS
 $dv + a + da$. AS BEFORE, THE SPEED OF
THE GAS BEHIND WAVE ② IS $2dv$;
THE PRESSURE AND TEMPERATURE LIKEWISE
INCREASE TO $p_2 + dp$, $p_2 + dp_2$ AND $T_0 + dt_1 + dt_2$.
THUS, SINCE WAVE ② IS MOVING FASTER
THAN WAVE ①, IT EVENTUALLY OVERTAKES
WAVE ①. WHEN THIS HAPPENS, THE
AND TEMPERATURE
PRESSURE \wedge JUMPS ACROSS THE LEADING
COALESCED WAVE INCREASE FROM
 dp_1 TO $dp_1 + dp_2$ AND FROM dt_1 TO
 $dt_1 + dt_2$. BY SIMILAR REASONING, WE CAN
SEE THAT EACH INCREMENTAL INCREASE
IN THE PISTON'S VELOCITY PRODUCES
A WAVE WHICH MOVES FASTER THAN ALL
PRECEDING WAVES. ALTHOUGH THE SPEED OF
THE COALESCED WAVE CONTINUES TO INCREASE
AS MORE TRAILING WAVES CATCH UP TO
IT, ITS ^{ABSOLUTE} SPEED IS NEVER GREATER
THAN THOSE OF THE TRAILING
(INFINITESIMAL) WAVES BEHIND IT,
(PROOF LATER.) THE COALESCED WAVE
THUS CONTINUES TO INCREASE IN
STRENGTH (AS MEASURED BY THE

2/3/00 (3)
 SIZE OF THE PRESSURE JUMP ACROSS IT) EVENTUALLY FORMING A (MOVING) NORMAL SHOCK.

B) STATIONARY NORMAL SHOCK RELATIONS

WE'LL FIRST DERIVE WORKING EQUATIONS FOR A NORMAL SHOCK THAT REMAINS IN A FIXED POSITION RELATIVE TO A NON-MOVING REFERENCE FRAME.

THE RESULTS CAN THEN BE EASILY ADAPTED TO THE CASE OF A MOVING NORMAL SHOCK - WE'LL SIMPLY FIX A COORDINATE SYSTEM TO THE MOVING SHOCK AND CAREFULLY BOOK KEEP THE UPSTREAM AND DOWNSTREAM GAS SPEEDS ACROSS THE SHOCK. LATER, THESE SAME RELATIONS (FOR STATIONARY NORMAL SHOCKS) WILL BE ADAPTED TO PROBLEMS INVOLVING OBLIQUE SHOCKS.

PHYSICAL FEATURES OF NORMAL SHOCK FLOW

$T_{02} = T_{01}$	$M_2 < 1$		$\leftarrow u_1$	$M_1 > 1$
$P_{02} < P_{01}$	$u_2 < u_1$		P_2	P_1
	$P_2 > P_1$		T_2	T_1
	$T_2 > T_1$		ρ_2	ρ_1
	$\rho_2 > \rho_1$		s_2	s_1
	$s_2 > s_1$			

IN NORMAL SHOCK FLOW, THE VELOCITY UPSTREAM IS PERPENDICULAR TO THE SHOCK, & DOWNSTREAM TO THE SHOCK.

ASSUMPTIONS

- 1) CALORICALLY PERFECT GAS =>
 - $C_p = \text{const}$
 - $C_v = \text{"}$
 - $e = C_v T$
 - $h = C_p T$
 - $p = \rho R T$
- 2) STEADY STATE CONDITIONS
- 3) UNIFORM PROPERTIES UPSTREAM & DOWNSTREAM OF SHOCK
- 4) NO DISSOCIATION OR IONIZATION ACROSS SHOCK

OBJECTIVE DERIVE ^{WORKING} RELATIONSHIPS GIVING

$\frac{p_2}{p_1}$, $\frac{\rho_2}{\rho_1}$, $\frac{T_2}{T_1}$, $\frac{p_{02}}{p_{01}}$, AND M_2 EACH AS A FN. OF THE UPSTREAM MACH NO. M_1 .

APPROACH MANIPULATE CONTIN, CONS. OF MASS AND CONS. OF ENERGY ETC., INTO FORMS INVOLVING $\frac{p_2}{p_1}$, $\frac{\rho_2}{\rho_1}$ AND $f_0(M_1, M_2)$; ELIMINATE $\frac{p_2}{p_1}$, $\frac{\rho_2}{\rho_1}$, AND SOLVE FOR M_2 IN TERMS OF M_1 ; BALANCE DESIRED EONS. CONS. OF MASS (ACROSS SHOCK);

$$\rho_1 u_1 = \rho_2 u_2$$

DIVIDE BY u_1 :

$$\rho_1 \frac{u_1}{u_1} = \rho_2 \frac{u_2}{u_1} = \rho_2 \frac{u_2}{u_2} \frac{u_2}{u_1} = \left(\rho_2 M_2 \left(\frac{u_2}{a_1} \right) \right) = \rho_1 M_1$$

$$\Rightarrow \boxed{\frac{p_2}{p_1} = \left(\frac{m_1}{m_2}\right) \left(\frac{a_1}{a_2}\right)} \quad (1)$$

CONS. OF MOMENTUM :

$$p_1 + p_1 u_1^2 = p_2 + p_2 u_2^2 \quad (2)$$

$$\Rightarrow \text{but } p = \rho RT$$

$$p = \frac{\rho}{\gamma} a^2 \quad (3)$$

$$(3) + (2) \Rightarrow \frac{p_1 a_1^2}{\gamma} + p_1 u_1^2 = \frac{p_2 a_2^2}{\gamma} + p_2 u_2^2 \quad (4)$$

$$(4) / a_1^2 \Rightarrow p_1 + \gamma p_1 M_1^2 = p_2 \left(\frac{a_2}{a_1}\right)^2 + \gamma p_2 M_2^2 \left(\frac{a_2}{a_1}\right)^2 \quad (5)$$

$$\Rightarrow p_1 (1 + \gamma M_1^2) = p_2 \left(\frac{a_2}{a_1}\right)^2 (1 + \gamma M_2^2)$$

$$\Rightarrow \boxed{\frac{p_2}{p_1} = \left(\frac{a_1}{a_2}\right)^2 \left(\frac{1 + \gamma M_1^2}{1 + \gamma M_2^2}\right)} \quad (6)$$

CONS. OF ENERGY :

$$h_1 + \frac{1}{2} u_1^2 = h_2 + \frac{1}{2} u_2^2$$

$$c_p T_1 + \frac{1}{2} u_1^2 = c_p T_2 + \frac{1}{2} u_2^2$$

$$\frac{\gamma R T_1}{\gamma - 1} + \frac{1}{2} u_1^2 = \frac{\gamma R T_2}{\gamma - 1} + \frac{1}{2} u_2^2$$

DIVIDE

CASE CAN

BY a_1^2 :

$$\frac{1}{\gamma - 1} + \frac{1}{2} M_1^2 = \left(\frac{a_2}{a_1}\right)^2 \left(\frac{1}{\gamma - 1}\right) + \frac{1}{2} M_2^2 \left(\frac{a_2}{a_1}\right)^2 \quad (7)$$

2/3/00 (7)

$$\Rightarrow \left(\frac{a_2}{a_1} \right)^2 = \frac{1 + \left(\frac{\delta-1}{2}\right)M_1^2}{1 + \left(\frac{\delta-1}{2}\right)M_2^2} \quad (8)$$

NOW eliminate $\left(\frac{a_2}{a_1}\right)$ FROM (1) & (6);

$$(1) \& (6) \Rightarrow \left(\frac{M_1}{M_2}\right) \left(\frac{a_1}{a_2}\right) = \left(\frac{a_1}{a_2}\right)^2 \left(\frac{1 + \delta M_1^2}{1 + \delta M_2^2}\right) \quad (9)$$

$$(9) \Rightarrow \frac{a_1}{a_2} = \left(\frac{M_1}{M_2}\right) \left(\frac{1 + \delta M_2^2}{1 + \delta M_1^2}\right) \quad (10)$$

SQUARE (10) AND ELIMINATE $\left(\frac{a_1}{a_2}\right)$ FROM THE RESULTING EQN AND (8);

$$\left(\frac{M_1}{M_2}\right)^2 \left(\frac{1 + \delta M_2^2}{1 + \delta M_1^2}\right)^2 = \frac{2 + (\delta-1)M_2^2}{2 + (\delta-1)M_1^2} \quad (11)$$

FOLLOWING SOME ALGEBRA (SEE ATTACHMENT), THIS CAN BE SOLVED FOR M_2^2 IN TERMS OF M_1^2 ;

$$M_2^2 = \frac{[M_1^2 + \left(\frac{2}{\delta-1}\right)]}{\left[\left(\frac{2\delta}{\delta-1}\right)M_1^2 - 1\right]} \quad (12)$$

2/3/00 (8)

PROOF THAT (2) FOLLOWS FROM (1);

FROM (1):

$$(2m_1^2 + (\delta-1)m_1^4)(1 + 2\delta m_2^2 + \delta^2 m_2^4)$$

$$(2m_2^2 + (\delta-1)m_2^4)(1 + 2\delta m_1^2 + \delta^2 m_1^4)$$

$$\Rightarrow 2(m_1^2 - m_2^2) + (\delta-1)(m_1^4 - m_2^4) + 2\delta(\delta-1)(m_2^2 m_1^4 - m_1^2 m_2^4) = 0$$

$$\text{or } 2(m_1^2 - m_2^2) + (\delta-1)(m_1^2 + m_2^2)(m_1^2 - m_2^2) + (\delta-1)2\delta m_1^2 m_2^2 (m_1^2 - m_2^2) = 0$$

OR
SINCE
 $m_1 \neq m_2$

$$2 + (\delta-1)(m_1^2 + m_2^2) + (\delta-1)2\delta m_1^2 m_2^2 = 0$$

$$\Rightarrow m_2^2 [(\delta-1) + (\delta-1)2\delta m_1^2] = (1-\delta)m_1^2 - 2$$

$$\rightarrow m_2^2 = \frac{m_1^2 - \frac{2}{(1-\delta)}}{1 + 2\delta m_1^2} = \frac{(1-\delta)m_1^2 - 2}{(1-\delta) + 2\delta m_1^2}$$

$$\boxed{m_2^2 = \frac{(\delta-1)m_1^2 + 2}{2\delta m_1^2 - (\delta-1)}}$$

END OF PROOF

2/3/00 (9)

NOW USE (12) IN (8) TO GET T_2/T_1 :

$$(8) + (12) \Rightarrow \left(\frac{a_2^2}{a_1^2} \right) = \frac{T_2}{T_1} = \frac{1 + (\gamma-1)M_1^2}{1 + (\frac{\gamma-1}{2})f(M_1^2)} \quad (13)$$

WHERE

$$M_2^2 = f(M_1^2) = \frac{(\gamma-1)M_1^2 + 2}{2\gamma M_1^2 - (\gamma-1)} \quad (14)$$

TO GET $\frac{P_2}{P_1}$, FIRST USE IDEAL GAS LAW TO GET:

$$\frac{P_2}{P_1} = \frac{P_2 R T_2}{P_1 R T_1} = \left(\frac{P_2}{P_1} \right) \left(\frac{a_2^2}{a_1^2} \right) \quad (15)$$

NOW USE (6) TO ELIMINATE $\frac{P_2}{P_1}$ FROM (15):

$$\frac{P_2}{P_1} = \frac{1 + \gamma M_1^2}{1 + \gamma M_2^2}$$

$$\text{OR } \frac{P_2}{P_1} = \frac{1 + \gamma M_1^2}{1 + \gamma f(M_1^2)} \quad (16)$$

THIS LAST EQN CAN BE MANIPULATED TO GIVE

$$\frac{P_2}{P_1} = \frac{2\gamma M_1^2 - (\gamma-1)}{\gamma+1} \quad (16a)$$

(NO PROOF)

2/3/00 (10)

Given $\frac{P_2}{P_1}$ (in eqn. (6)) and $\frac{T_2}{T_1}$ (eqn. (3)), we can now get $\frac{P_2}{P_1}$:

$$\frac{P_2}{P_1} = \frac{(P_2 / \rho_2)}{(P_1 / \rho_1)} = \left(\frac{P_2}{P_1} \right) \left(\frac{T_2}{T_1} \right)^{-1} \quad (17)$$

\uparrow \uparrow
 eqn (6) eqn (3)

It can be shown that (17) can be simplified to give:

$$\frac{P_2}{P_1} = \frac{(\gamma + 1) M_1^2}{2(\gamma - 1) M_1^2} \quad (17A)$$

(no proof)

To get the change in stagnation pressure across the shock (i.e.

$\frac{P_{02}}{P_{01}}$), recall from last lecture

that

$$\frac{P_0}{P} = \left(1 + \frac{(\gamma - 1)}{2} M^2 \right)^{\frac{\gamma}{\gamma - 1}} \quad (18)$$

THUS,

$$\frac{P_{02}}{P_{01}} = \left(\frac{P_{02}/P_2}{P_{01}/P_1} \right) \left(\frac{P_2}{P_1} \right)$$

FROM (18) AND (16), THIS BECOMES;

$$\frac{P_{02}}{P_{01}} = \left[\frac{\left(1 + \frac{\gamma-1}{2} \gamma M_1^2\right)}{\left(1 + \frac{\gamma-1}{2} \gamma M_2^2\right)} \right]^{\frac{\gamma}{\gamma-1}} \left[\frac{1 + \gamma M_2^2}{1 + \gamma M_1^2} \right] \quad (19)$$

* NOTE TABLE A.2 CAN BE USED ONLY FOR AIR
AIR (WHERE $\gamma = 1.4$)
BEFORE LOOKING AT A FEW EXAMPLES.

IT IS PHYSICALLY ENLIGHTENING
TO LOOK AT WHAT HAPPENS TO

THE RATIOS OF $\frac{P_2}{P_1}$, $\frac{T_2}{T_1}$, $\frac{P_0}{P_1}$ AND $\frac{P_{02}}{P_{01}}$

AND TO M_2 AS M_1 BECOMES LARGE
(i.e., $M_1 \rightarrow \infty$). LET'S LIMIT THEM TO AIR
(WHERE $\gamma = 1.4$):

FROM (12):

$$M_2^2 (M_1^2 \rightarrow \infty) \rightarrow \frac{\gamma-1}{2\gamma} = \frac{1.4-1}{2 \cdot 1.4} = \frac{1}{7}$$

FOR AIR

FROM (16) [w/ $\gamma \rightarrow 1/4$]

$$\frac{P_2}{P_1} \rightarrow \frac{\gamma M_1^2}{1 + \gamma \left(\frac{1}{4}\right)} \rightarrow \infty \quad \text{AS } M_1 \rightarrow \infty$$

similarity, from (13):

2/3/00 (12)

$$\left[\frac{T_2}{T_1} \rightarrow \frac{(\gamma-1) M_2^2}{1 + (\gamma-1) \left(\frac{1}{2}\right)} \rightarrow \infty \right] \text{ as } M_1 \rightarrow \infty$$

Finding 1, from (17A):

$$\left[\frac{P_2}{P_1} \rightarrow \frac{(\gamma+1)}{(\gamma-1)} = \frac{2.4}{.4} = 6 \right] \text{ as } M_1 \rightarrow \infty$$

IN ORDER TO SHOW THAT THE STAGNATION TEMP TO REMAINS CONSTANT NEARBY A NORMAL SHOCK, LOOK AT THE ENERGY EQU. ACROSS THE SHOCK;

BY DEFN., THE STAGNATION ENTHALPY, h_0 , IS THE ENTHALPY. EXTANT WHEN A FLOW IS ACCELERATED OR DECELERATED ISENTROPICALLY FROM OR TO A SPEED u AND TEMP, T , THUS, FOR 1-D ^{STEADY} FLOW, AN ENERGY BALANCE (IN WHICH FRICTIONAL WORK AND HEAT TRANSFER ARE BY ASSUMPTION NEGLECTED) BETWEEN A STAGNATION POINT AND AN ARBITRARY UPSTREAM (OR DOWNSTREAM) LOCATION (WHERE THE

2/3/00 (13)

gas speed is u and temp. is T)

yields:

$$\left[\frac{1}{2} u^2 + c_p T = c_p T_0 \right] \quad (20)$$

now, across a normal shock, the energy is conserved

$$\frac{1}{2} u_1^2 + c_p T_1 = \frac{1}{2} u_2^2 + c_p T_2 \quad (21)$$

but, by definition the LHS of (21) equals the stagnation enthalpy $h_{01} = c_p T_{01}$ while the RHS equals $h_{02} = c_p T_{02}$.
(i.e., at any point where the flow is the local speed is u and the temperature is T , we can always associate a stagnation enthalpy.)
(by imagining an isentropic deceleration)

thus (21) implies that

$$c_p T_{01} = c_p T_{02}$$

or $\boxed{T_{01} = T_{02}} \quad (22)$

2/3/20 (14)

EXAMPLE 1

A gas w/ molar mass 28.9 (g/g.mol) and specific heat ratio of 1.4 discharges through a nozzle. A normal shock wave occurs @ a section where the Mach no. is 2.5, pressure is 40 kPa and temp is -20°C .

Find downstream Mach no., pressure and temperature,

$M_1 = 2.5$	$M_2 ?$
$P_1 = 40 \text{ kPa}$	$P_2 ?$
$T_1 = 253 \text{ K}$	$T_2 ?$

$$\gamma = 1.4$$

$$R = 8.314 \text{ J/gmol}\cdot\text{K} \cdot \left(\frac{\text{gmol}}{28.9 \text{ g}} \right) \cdot \frac{10^3 \text{ g}}{\text{kg}} = 287.7 \text{ J/kg}\cdot\text{K}$$

Let's use TABLE A.2 to solve:

AT $M_1 = 2.5$; TABLE A.2 gives

$$\frac{P_2}{P_1} = 7.125, \quad \frac{T_2}{T_1} = 2.137$$

$$M_2 = 0.513$$

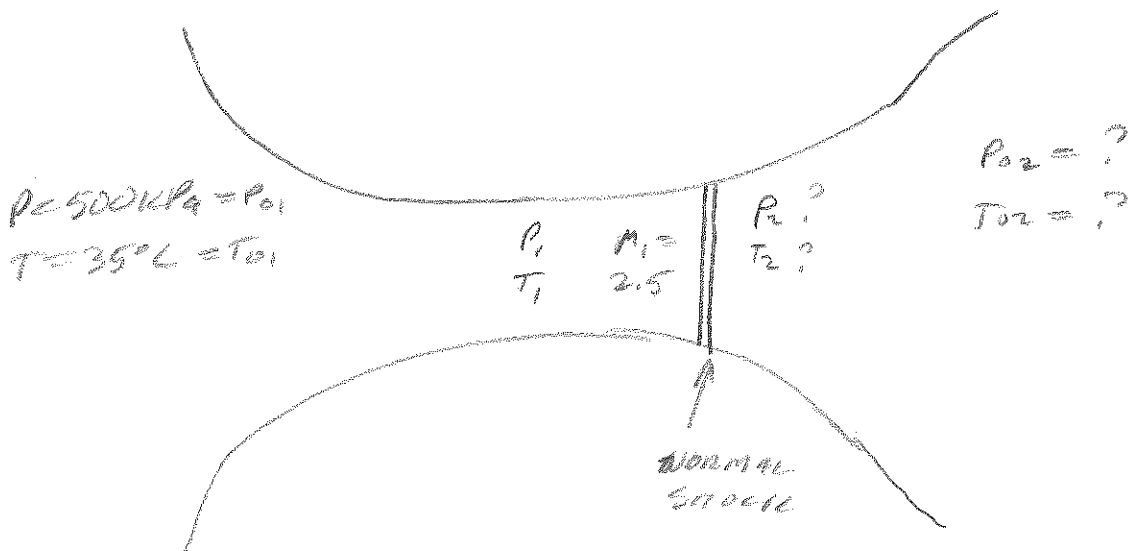
$$\Rightarrow \begin{cases} P_2 = (7.125)(40) = 285 \text{ kPa} \\ T_2 = (2.137)(253) = 540 \text{ K} \end{cases}$$

02/3/00

(15)

EXAMPLE 2

AIR IS EXPANDED FROM A LARGE RESERVOIR IN WHICH PRESS. AND TEMP. ARE 500 kPa AND 35°C THROUGH A VARIABLE AREA DUCT. A NORMAL SHOCK OCCURS AT A POINT WHERE THE MACH NO. IS 2.5. WHAT'S THE PRESSURE AND TEMP. JUST DOWNSTREAM OF THE SHOCK? DOWNSTREAM OF THE SHOCK, THE FLOW IS BROUGHT TO REST IN ANOTHER RESERVOIR. WHAT'S THE PRESSURE AND TEMP. IN THIS DOWNSTREAM RESERVOIR? ASSUME ISENTROPIC 1-D FLOW.



2/03/00 (16)

AGAIN, LET'S USE TABLE A.2:

ITERATIVE STRATEGY:

- 1) USE ISENTROPIC RELATIONS TO GET P_1 & T_1 (GIVEN P_{01} , T_{01} , & M_1).
- 2) USE NORMAL SHOCK RELNS TO GET P_2 , T_2 (GIVEN T_1 , T_1).
- 3) USE NORMAL SHOCK RELNS TO GET P_{02} (GIVEN P_{01}).

FIRST, WE KNOW THAT

$$T_{02} = T_{01} = 308 \text{ K}$$

FROM TABLE A.1 (WHICH AGAIN, CAN ONLY BE USED FOR AIR (SINCE $\gamma = 1.4$))

@ $M_1 = 2.5$:

$$\frac{P_{01}}{P_1} = 17.09$$

$$\frac{T_{01}}{T_1} = 2.25$$

$$\Rightarrow P_1 = \frac{P_{01}}{17.09} = \frac{500 \text{ kPa}}{17.09} = 29.26 \text{ kPa}$$

$$T_1 = \frac{T_{01}}{2.25} = \frac{308 \text{ K}}{2.25} = 136.9 \text{ K}$$

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PAGE 17 FOR
ANSWERS

2/03/00

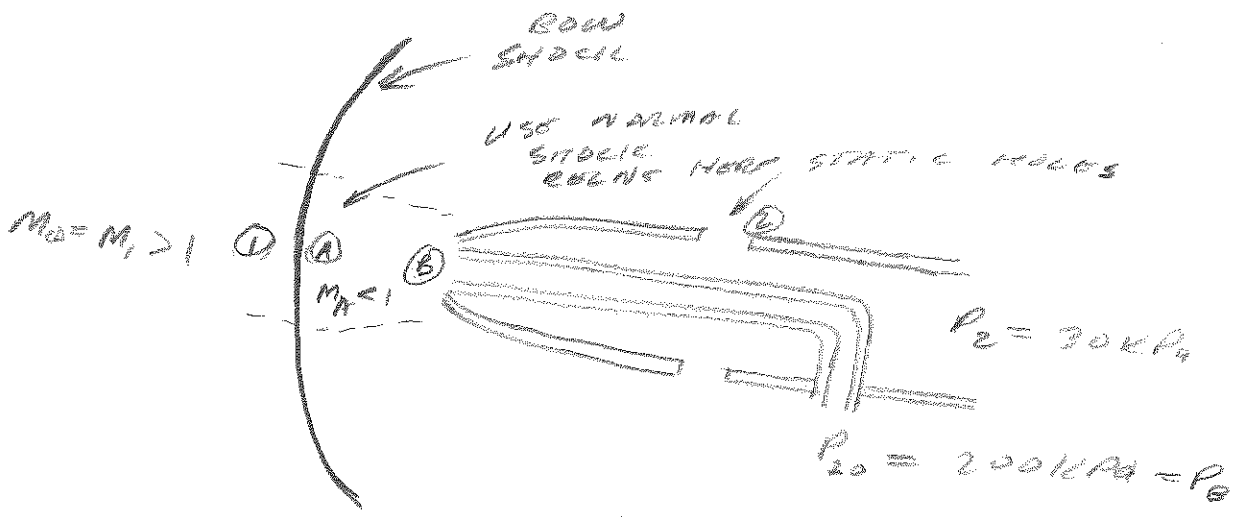
(16A)

EXAMPLE 3 PITOT TUBE IN SUPersonic FLOW

A PITOT TUBE (SHOWN) IS USED TO MEASURE THE FREE-STREAM MACH NO., M_∞ . THE PRESSURES MEASURED ARE SHOWN. IT IS KNOWN EXPERIMENTALLY THAT FOR THIS PITOT TUBE THE PRESSURE COEFFICIENT, C_p , DEFINED AS

$$C_p = \frac{P_2 - P_\infty}{\frac{1}{2} \rho_\infty U_\infty^2}$$

IS EQUAL TO 0.1. WHAT IS THE FREE STREAM MACH NO.?



ANSWER

FOR PITOT TUBE PROBLEMS, WE USE THE NORMAL SHOCK RELATIONS TO ESTIMATE PROPERTY CHANGES ACROSS THE LEADING PORTION OF THE BOW SHOCK. WE ALSO ASSUME THAT SINCE THE FLOW BEHIND THIS PART OF THE SHOCK IS

IS SUBSONIC THAT THE DECELERATION FROM PT. (A) TO PT. (B) IS ISENTROPIC.

LET'S REWRITE C_p AS

$$C_p = \frac{P_2 - P_0}{\frac{\rho P_0 U_0^2}{2 \gamma P_0}} = \left(\frac{2}{\gamma}\right) \left(\frac{1}{M_0^2}\right) \left(\frac{P_2}{P_0} - 1\right)$$

* NOTE P_2 IS THE ^{STATIC} P PRESSURE AT THE STATIC PRESSURE POINTS.

TRY TO COME UP W/ A SOLID APPROACH!

1) $\frac{P_{20}^{\checkmark}}{P_2^{\checkmark}} = f(M_2)$ = isentropic relation where M_2 is Mach No. @ PT. (2)

⇒ CAN BACK OUT M_2 (ASSUMES ISENTROPIC FLOW FROM PT (2) TO (2))

2) $\frac{P_A}{P_{20}} = f(M_A)$ = isentropic relation

3) $\frac{P_A}{P_2} = \left(\frac{P_A}{P_{20}}\right) \left(\frac{P_{20}}{P_2}\right) = f(M_A) f(M_2)$ (REARRANGE w/ 2)

4) $M_A = g(M_0)$ = eqn (2) = normal shock reln. between M_A and M_0

5) 3) + 4) ⇒ $\frac{P_A}{P_2} = f^{-1}(g(M_0)) f(M_2)$

⇒
$$P_A = P_2 f^{-1}(g(M_0)) f(M_2)$$

6) $\frac{P_A}{P_{20}} = g_f(M_0)$ = normal shock reln. between P_A & P_{20}

$$7) \quad 5) + 6) \Rightarrow P_0 = \frac{P_A}{g_P(M_0)}$$

$$P_0 = \frac{P_A \rho^{-1}(g(M_0)) f(M_0)}{g_P(M_0)}$$

8)

$$C_p = \frac{2}{\gamma} \left(\frac{1}{M_0^2} \right) \left[\frac{P_A}{\rho \left(\frac{P_A \rho^{-1}(g(M_0)) f(M_0)}{g_P(M_0)} \right)^{\frac{\gamma}{\gamma-1}}} - 1 \right]$$

⊛

⇒ solve 8) for $M_0 \Rightarrow$ soln

now list the functions f , g , & g_P :

$$\frac{P_0}{P} = f(M) = \left(\frac{\frac{1}{\gamma-1} + \frac{1}{2} M^2}{\frac{1}{\gamma-1}} \right)^{\frac{\gamma}{\gamma-1}} = \text{ISENTROPIC RELATION W/ } M_0 = 0$$

$$M_A = g(M_0) = \frac{\left(M_0^2 + \frac{2}{\gamma-1} \right)}{\left(\frac{2\gamma}{\gamma-1} M_0^2 - 1 \right)} = \text{NORMAL SHOCK RELATION (EQU (12))}$$

$$\frac{P_A}{P_0} = g_P(M_0) = \frac{2\gamma M_0^2 - (\gamma-1)}{\gamma+1} = \text{NORMAL SHOCK RORN. (EQU. 16A)}$$

⊛
SOLVE BY ITERATION - YOU SHOULD GET FOLLOWING:

$$M_0 = 1.93$$

2/03/00 (17)

now from TABLE A.2:

② $M_1 = 2.5$; $\frac{P_2}{P_1} = 7.125$ $\frac{T_2}{T_1} = 2.137$ $\frac{P_{02}}{P_{01}} = 0.499$

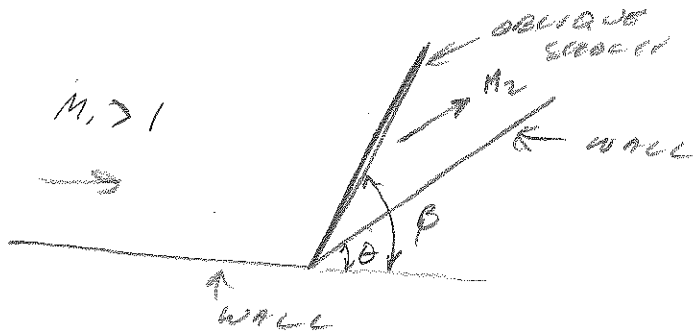
\Rightarrow

$P_2 = (7.125)(29.26) = 208.5 \text{ kPa}$
$T_2 = (2.137)(136.9) = 292.6 \text{ K}$
$P_{02} = (0.499)(500) = 250 \text{ kPa}$

OBlique SHOCKS

EXAMPLE

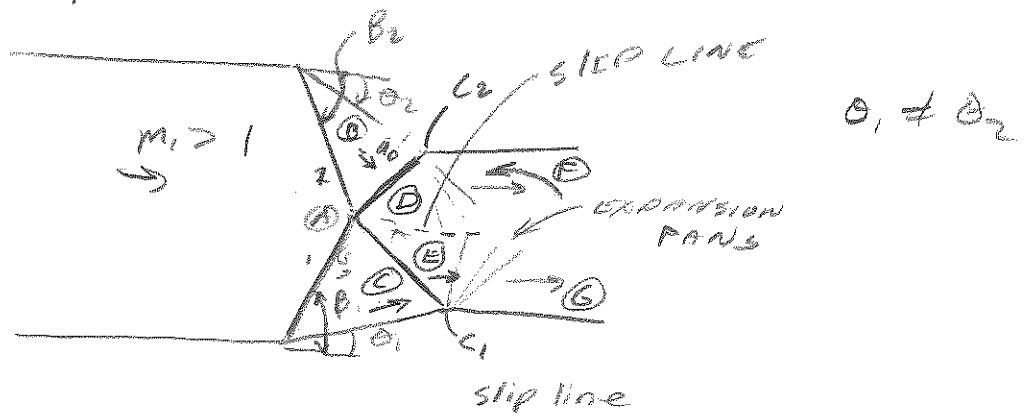
SUPERSONIC FLOW INTO A
COMPRESSIVE CORNER



ONCE FLOW PASSES THROUGH THE OBLIQUE SHOCK, IT HAS TURNED UPWARD BY AN ANGLE θ .

2/03/00 (18)

EXAMPLE 2



IN THIS FLOW, THE SUPersonic FLOW
PASSES INTO TWO OPPOSING COMPRESSING
CORNERS (= CORNERS WHICH TURN
THE FLOW INTO ITSELF -- I.E., CORNERS
HAVING AN INCLUDED ANGLE LESS THAN
(θ_1 & θ_2 IN THIS CASE)
 180°). TWO OBLIQUE SHOCKS (1 & 2)
FORM AT ANGLES β_1 & β_2 RELATIVE
TO HORIZONTAL. THE SHOCKS MEET
AT PT. (A) WHERE, DUE TO THE FACT THAT
THE FLOWS IN REGIONS (B) & (C) ARE
AGAIN TURNED COMPRESSIVELY (BY
A SLIP-LINE EMANATING FROM PT (A)),
TWO MORE OBLIQUE SHOCKS FORM.
THE FOUR SHOCKS FORM AN 'X'
PATTERN W/ CORNER AT PT. (A).
THE FLOW IN REGION (B) IS PARALLEL
TO THE UPPER INCLINED WALL; SIMILARLY,
FLOW IN REGION (C) IS PARALLEL TO THE

2/03/00

(19)

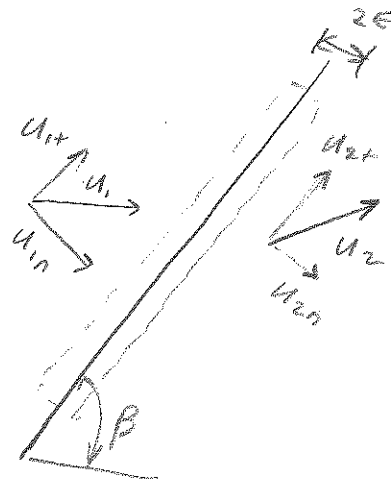
LOWER INCLINED WALL. THE SECOND
SET OF OBLIQUE SHOCKS (WHICH NOMINALLY
WENT AT THE CORNERS C_1 AND C_2)
FORM BECAUSE THE SLIP-LINE
BEHAVES ESSENTIALLY LIKE A VIRTUAL
WALL W/IN THE FLOW, FORCING THE
FLOW LEAVING THE SECOND SET OF
SHOCKS TO FLOW PARALLEL TO THE
SLIP-LINE (IN REGIONS (D) & (E)).
FINALLY, THE FLOWS IN (D) & (E) ARE
TURNED THROUGH THE EXPANSION
FANS EMANATING FROM C_1 & C_2 —
ONLY THE FLOWS PASS THROUGH
THE FANS, THEIR DIRECTIONS ARE
PARALLEL TO THE DOWNSTREAM (PARALLEL)
OUT WALLS. WE WILL SOON BE
ABLE TO ANALYZE THIS FLOW COMPLETELY

2/03/00 (20)

B) DERIVATION OF WORKING EQUATIONS

$M_1 > 1$

- M_1
- T_1
- P_1
- E_1



APPROACH: SHOW THAT TANGENTIAL VELOCITY COMPONENTS (u_{1t} & u_{2t}) ARE EQUAL (USING CONSERV. OF TANGENTIAL MOMENTUM), THEN SHOW THAT CONSERV. OF MASS, MOMENTUM (NORMAL COMPONENT), AND ENERGY HAVE THE SAME FORM AS THE " DERIVED FOR NORMAL SHOCKS. HERE, HOWEVER, THE NORMAL VELOCITY COMPONENTS (u_{1n} & u_{2n}) REPLACES THE TOTAL VELOCITIES (u_1 & u_2) USED IN DERIVING THE NORMAL SHOCK RELATIONS. ONCE THIS IS SHOWN, THEN ALL OF THE " PREVIOUSLY DERIVED SHOCK EONS. CAN BE WRITTEN IN TERMS OF u_{1n} & u_{2n} . - THAT COMPLETES THE DERIVATION.

TO PROVE THAT $u_H = u_{2t}$, APPLY
 CONG. OF LINEAR MOM. IN TANGENTIAL
 DIRECTION;

$$\begin{aligned} \int_S \rho u_i (u \cdot \hat{n}) dS &= \int_S \rho_{ij} n_j dS && (S = \text{surface surrounding CV}) \\ &= \int_S -p n_i dS && (\text{Choose } i=1 \text{ direction to be the tangent direction}) \\ &= -P_{top}(2\epsilon)b + P_{bottom}(2\epsilon)b \end{aligned}$$

$b = \text{width of control volume}$
 (into page)

$$\begin{aligned} \int_S \rho u_t (u \cdot \hat{n}) dS &= \int_{S_1} \rho u_t (u \cdot \hat{n}) dS + \int_{S_2} \rho u_t (u \cdot \hat{n}) dS \\ &\downarrow \text{use } u \cdot \hat{n} \text{ flow assumption} \\ &= -\rho_1 u_H u_{in} S_1 + \rho_2 u_H u_{in} S_2 \end{aligned}$$

NOW SINCE $S_1 \approx b w$ AND $S_2 \approx b w$ (i.e., THE TANGENTIAL LENGTH OF THE CONTROL VOLUME w IS MUCH GREATER THAN THE THICKNESS ϵ), THEN PRESSURE TERMS ARE NEGLIGIBLE AND MOMENTUM CAN SIMPLIFY TO

$$\boxed{\rho_1 u_H u_{in} S_1 = \rho_2 u_H u_{in} S_2} \quad (23)$$

NOW, WE CAN APPLY CONE. OF MASS TO SHOW THAT $\rho_1 u_{1n} = \rho_2 u_{2n}$

$$\frac{dM_{CV}}{dt} = \oint \rho(\mathbf{u} \cdot \mathbf{n}) ds = 0$$

$$\Rightarrow \int_{S_1} \rho(\mathbf{u} \cdot \mathbf{n}) ds + \int_{S_2} \rho(\mathbf{u} \cdot \mathbf{n}) ds = 0$$

USE
UNIFORM
FLOW
ASSUMPTION \rightarrow

$$-\rho_1 u_{1n} S_1 + \rho_2 u_{2n} S_2 = 0$$

$$\Rightarrow \boxed{\rho_1 u_{1n} = \rho_2 u_{2n}}$$

(24)

THUS, USING (24) IN (23), WE FINALLY SHOW THAT

$$\boxed{u_{1t} = u_{2t}}$$

(25)

TO GET THE THIRD NEEDED EQN., APPLY CONE. OF LIN. MOMENTUM IN THE NORMAL DIRECTION:

$$\begin{aligned} \frac{d}{dt} \int_{CV} \rho \mathbf{u} dV &= \frac{d}{dt} \int_{CV} \rho \mathbf{u} dV + \oint_S \rho \mathbf{u} (\mathbf{u} \cdot \mathbf{n}) ds \\ &= \Sigma F_n \\ &= \oint_S \sigma_{ij} n_j ds \end{aligned}$$

let index 1 correspond to

normal direction

INDIVIDUALLY

CONSIDER EACH TERM IN LAST EQN.:

$$\int_S \rho u_n(u \cdot a) ds = \int_{S_1} \rho u_n(u/a) ds + \int_{S_2} \rho u_n(u/a) ds$$

(UNIF. FLOW ASSUMPTION) $\Rightarrow -\rho_1 u_{1n}^2 S_1 + \rho_2 u_{2n}^2 S_2$

$$\int_S \rho u_n n_j ds = \int_S -P_{n_j} ds = - \int_{S_1} P_{n_j} ds - \int_{S_2} P_{n_j} ds \quad (1)$$

(neglect viscous terms in σ_{ij}) $= P_1 S_1 - P_2 S_2$

THUS,

$$\rho_2 u_{2n}^2 + P_2 = \rho_1 u_{1n}^2 + P_1 \quad (26)$$

THE FINAL EQN. WE NEED IS THE ENERGY BALANCE. USING THE SAME APPROACH AS ABOVE, WE FIND THAT

$$c_p T_1 + \frac{u_{1n}^2}{2} = c_p T_2 + \frac{u_{2n}^2}{2} \quad (27)$$

NOW

A. COMPARING EQNS. (24), (26) & (27) WITH THE BALANCE EQNS. OBTAINED ACROSS A NORMAL SHOCK, WE SEE THAT THE 1st AND 2nd ARE OF IDENTICAL FORM W/ u_{1n} REPLACING u_1 , & u_{2n} REPLACING u_2 . THUS, IF WE DEFINE M_{n1} AND M_{n2} AS

$M_{n1} = \frac{u_{1n}}{a_1}$	(28)
$M_{n2} = \frac{u_{2n}}{a_2}$	(29)

WE OBTAIN

FROM (27)

$M_{n2}^2 = \frac{[M_{n1}^2 + \left(\frac{2}{\delta-1}\right)]}{\left[\left(\frac{2\delta}{\delta-1}\right)M_{n1}^2 - 1\right]}$	(30)
--	------

(OBLIQUE SHOCK)

FROM (16A)

$\frac{P_2}{P_1} = \frac{2\delta M_{n1}^2 - (\delta-1)}{(\delta+1)}$	(31)
--	------

(OBLIQUE SHOCK)

(B) ⇒

$$\frac{r_2}{r_1} = \frac{1 + \left(\frac{\sigma-1}{2}\right) M_{in}^2}{1 + \left(\frac{\sigma-1}{2}\right) f(M_{in}^2)}$$

$$f(M_{in}^2) = \frac{(\sigma-1) M_{in}^2 + 2}{2\sigma M_{in}^2 - (\sigma-1)}$$

(32)

(33)

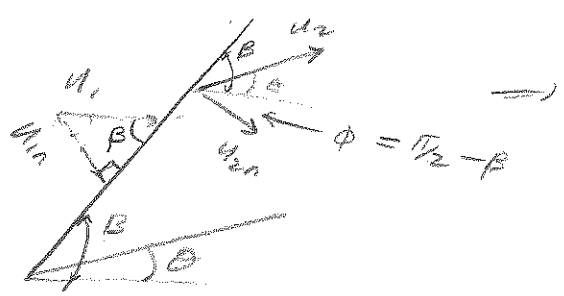
SHOCK
STOCK

(17A) ⇒

$$\frac{p_2}{p_1} = \frac{(\sigma+1) M_{in}^2}{2 + (\sigma-1) M_{in}^2}$$

(39)

IN ORDER TO COMPLETE THE DERIVATION WE NEED TO ROTATE u_{n1} & u_{n2} TO THE ANGLES β (SHOCK ANGLE) AND θ (TURNING ANGLE);



FROM FIGURE

$$u_{n1} = u_1 \sin \beta \quad (35)$$

$$u_{n2} = u_2 \cos(\phi + \theta) = u_2 \cos\left(\frac{\pi}{2} - \beta + \theta\right)$$

$$u_{n2} = u_2 \sin(\beta - \theta) \quad (36)$$

Thus

$$M_{1n} = \frac{u_{1n}}{a_1} = \frac{u_1}{a_1} \sin \beta = M_1 \sin \beta \quad (37)$$

$$M_{2n} = \frac{u_{2n}}{a_2} = \frac{u_2}{a_2} \sin(\beta - \theta) = M_2 \sin(\beta - \theta) \quad (38)$$

SO FINAL WORKING EQUATIONS ARE OBTAINED BY SUBSTITUTING (37) & (38) INTO (30), (31), (32), (33) & (34);

(30) \Rightarrow

$$M_2^2 \sin^2(\beta - \theta) = \frac{(M_1^2 \sin^2 \beta + \frac{2}{\gamma - 1})}{[\frac{2\gamma}{\gamma - 1} M_1^2 \sin^2 \beta - 1]} \quad (39)$$

(31) \Rightarrow

$$\frac{P_2}{P_1} = \frac{2\gamma M_1^2 \sin^2 \beta - (\gamma - 1)}{\gamma + 1} \quad (40)$$

(32) \Rightarrow

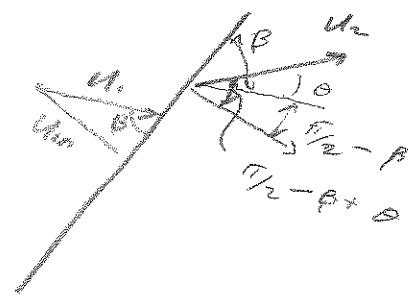
$$\frac{T_2}{T_1} = \frac{1 + \frac{(\gamma - 1)}{2} M_1^2 \sin^2 \beta}{1 + \frac{(\gamma - 1)}{2} \gamma (M_1^2 \sin^2 \beta)} \quad (41)$$

(34) \Rightarrow

$$\frac{P_2}{P_1} = \frac{(\gamma + 1) (M_1^2) \sin^2 \beta}{2 + (\gamma - 1) M_1^2 \sin^2 \beta} \quad (42)$$

IN ORDER TO SOLVE OBLIQUE SHOCK PROBLEMS, WE NEED A RELATIONSHIP BETWEEN θ , β AND M_1 . — M_1 & θ ARE USUALLY KNOWN AND, AS WE'LL SHOW, CAN thus BE USED TO DETERMINE THE SHOCK ANGLE β .

DERIVATION OF RELATIONSHIP BETWEEN θ , M , & β :



FROM FIG.

$$\frac{u_{1n}}{u_{1t}} = \tan \beta$$

$$\frac{u_{2n}}{u_{2t}} = \tan \left(\frac{\pi}{2} - (\frac{\pi}{2} - \beta + \theta) \right) = \tan(\beta - \theta)$$

BUT $u_{1t} = u_{2t}$ (AS SHOWN NEEDS)

ALSO, FROM CONTIN

$$\frac{u_{2n}}{u_{1n}} = \frac{\rho_1}{\rho_2} = \frac{u_{2n} u_{1t}}{u_{1n} u_{2t}} = \frac{\tan(\beta - \theta)}{\tan \beta}$$

OR USING (42) \Rightarrow

$$\boxed{\frac{\tan(\beta - \theta)}{\tan \beta} = \frac{2 + (\gamma - 1) M_1^2 \sin^2 \beta}{(\gamma + 1) M_1^2 \sin^2 \beta}} \quad (43)$$

NOW NOTE THAT

$$\begin{aligned} \tan(\beta - \theta) &= \frac{\tan \beta - \tan \theta}{1 + \tan \beta \tan \theta} \\ \Rightarrow \text{AS } \Rightarrow \quad 1 - \tan \theta (\tan \beta) &= \gamma (1 + \tan \beta \tan \theta) \\ &= \gamma + \gamma \tan \beta \tan \theta \end{aligned}$$

where
$$\xi = \frac{2 + (1 - M_1^2) \sin^2 \beta}{(1 + M_1^2) \sin^2 \beta} \quad (44A)$$

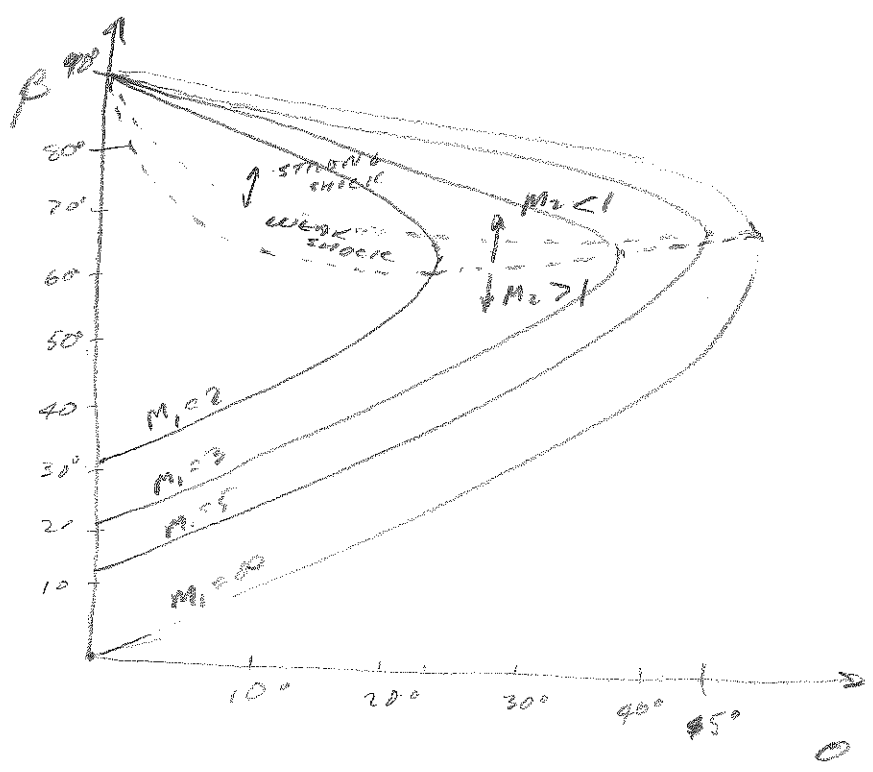
SOLVING LAST EQU ON PREVIOUS PAGE FOR $\tan \theta$ we get:

$$\tan \theta (-1 - \xi \tan^2 \beta) = \xi \tan \beta - \tan \beta$$

$$\tan \theta = \frac{\tan \beta (1 - \xi)}{(1 + \xi \tan^2 \beta)} \quad (44)$$

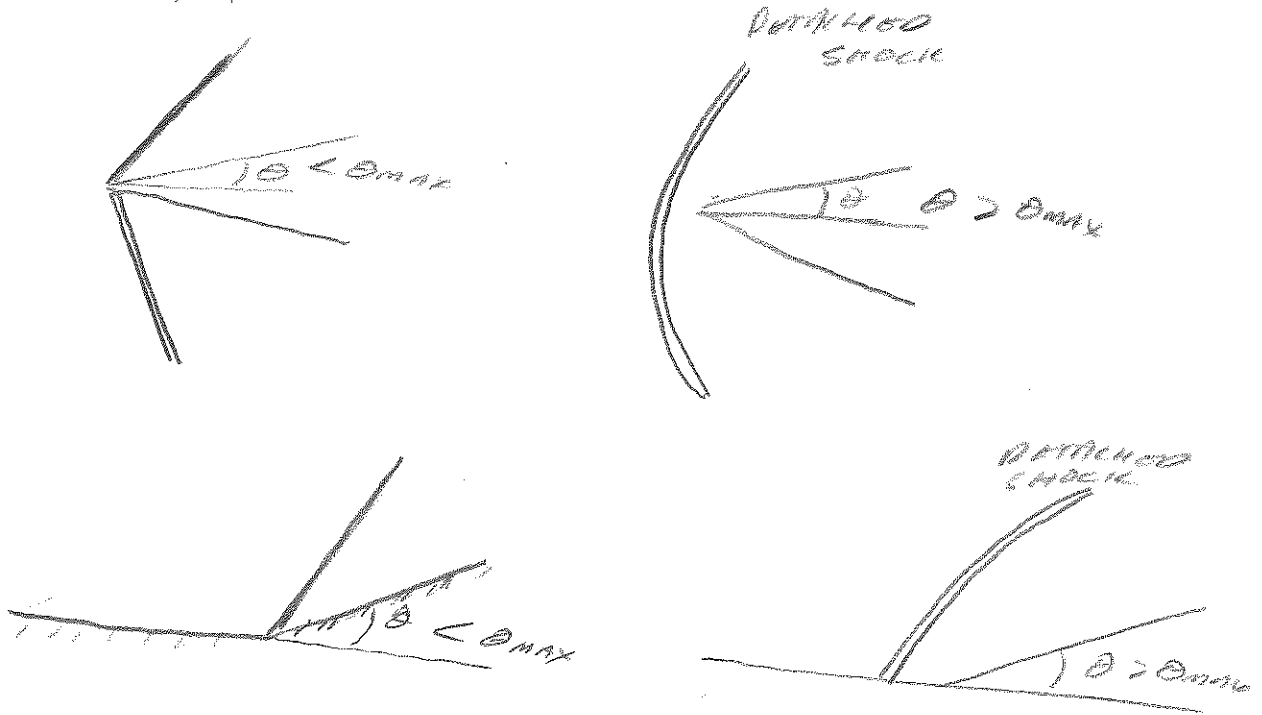
EQU (44) (44A) IS THE DESIRED RELATIONSHIP. IT GIVES θ AS A FN. OF M_1 AND β

AS DISCUSSED IN YOUR TEXT ON PGS 107-109, THERE ARE SEVERAL PHYSICAL IMPLICATIONS ASSOCIATED W/ EQU (44).



IMPLICATIONS OF EQN. (44)

1) FOR ANY GIVEN M_1 , THERE'S A MAXIMUM VALUE OF $\theta = \theta_{MAX}$ ALLOWING SOLN. OF (44). FOR DEFLECTION ANGLES $> \theta_{MAX}$, A STRAIGHT OBLIQUE SHOCK CANNOT EXIST (CORRESPONDING TO NO SOLN. OF (44)). FOR $\theta > \theta_{MAX}$, THE SHOCK ^{TURNS} MOVES AWAY FROM THE Δ SURFACE. EXAMPLES (FROM TEXT);



2) FOR ANY $\theta < \theta_{MAX}$ AND ANY GIVEN M_1 , THERE ARE TWO VALUES OF β THAT SATISFY (44). THE LARGER β CORRESPONDS TO A STRONG SHOCK WHILE THE SMALLER β GIVES THE WEAK SHOCK ANGLE. THE WEAK SHOCK SOLN. TEND TO OCCUR IN MOST CASES. INDEED THE WEAK

2/3/00

SHOCK SOLUTION IS FAVORED FROM AN ENTROPY STANDPOINT (THE ENTROPY CHANGE ACROSS A WEAK OBLIQUE SHOCK IS SMALLER THAN OVER A STRONG SHOCK). THE UPSTREAM MACH NO., M_1 , IS SUPERSONIC BEHIND A WEAK SHOCK (EXCEPT NEAR θ_{max} AS SHOWN IN FIG. 4.5 IN TEXT.), AND ALWAYS SUBSONIC BEHIND A STRONG SHOCK.

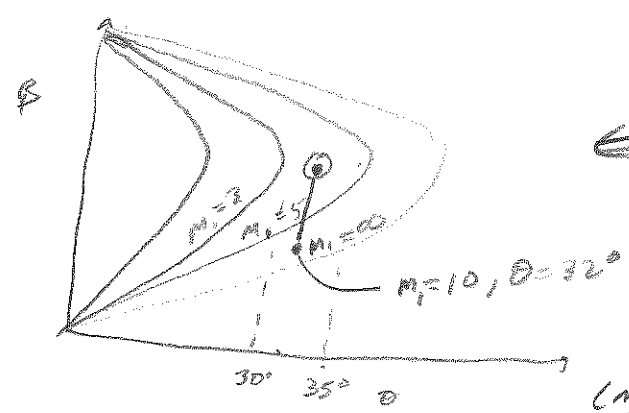
3) IF $\theta = 0^\circ$, (44) HAS SOLUTIONS $B = \frac{\pi}{2}$ (CORRESPONDING TO A STRONG SHOCK) AND

$B = \sin^{-1} \frac{1}{M_1} = \mu$. (WE WILL DISCUSS THE OTHER SOLUTION, NEXT LECTURE.)

4) FOR FIXED θ (DEFLECTION ANGLE), THE VALUE OF β INCREASES AS M_1 DECREASES. ONLY M_1 DROPS BELOW SOME CRITICAL VALUE, (44) HAS NO SOLUTION, AND THE ATTACHED OBLIQUE SHOCK DETACHES.

EXAMPLE

LET $\theta = 32^\circ$
 SHOW GRAPHICALLY THAT AS M_1 DECREASES FROM SAY 10, A CRITICAL M_1 IS EVENTUALLY REACHED WHERE NO SOLUTION EXISTS

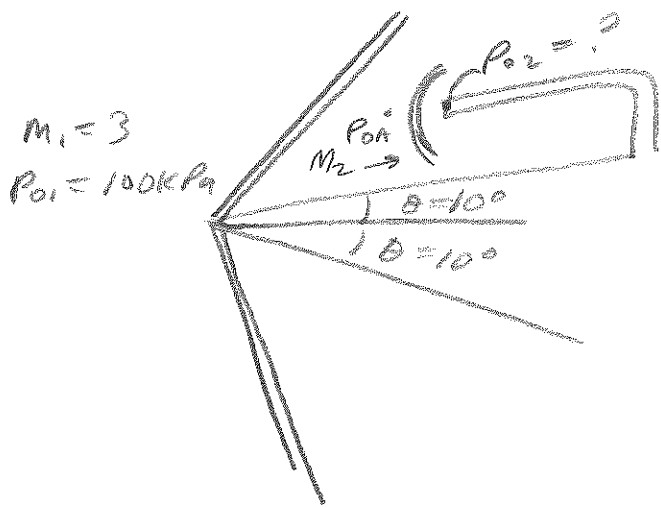


FROM GRAPH, SOMEWHERE BETWEEN $M_1 = 5$ AND $M_1 = 3$, A SOLUTION IS NO LONGER POSSIBLE

(NOTE $M_1=3$ CURVE NEVER INTERSECTS $\theta=32^\circ$)

EXAMPLE

A wedge is placed in a flow where $M_1 = 3$ and the free stream stagnation pressure is $P_{01} = 100 \text{ kPa}$. If a pitot tube is located behind the oblique shock as shown, what stagnation pressure will it read?



SOLN. STRATEGY

- 1) Given M_1 & θ , get β (assume weak shock)
from FIGURE.
- 2) Given β and M_1 , get $M_{2n} = f_n(M_{1n})$
($M_{1n} = M_1 \sin \beta$)
- 3) Given M_{2n} , get $M_2 = M_{2n} / \sin(\beta - \theta)$
- 4) Given M_2 , get $\frac{P_{02}}{P_{01}} = f_n(M_2) = \text{normal shock eqn. for } \frac{P_{02}}{P_{01}}$

2/3/00 (32)

5) $\frac{P_{0A}}{P_{01}} = f_n(M_{1A}) =$ normal shock eqn for $\frac{P_{02}}{P_{01}}$ in terms of M_{1A} .

$\rightarrow P_{0A} = P_{01} f_n(M_{1A})$

6) From 4) and 5) solve for P_{02} :

$$P_{02} = P_{0A} f_n(M_{2A}) = P_{01} f_n(M_{1A}) f_n(M_{2A})$$

Solve:

1) For $M_1 = 3$ $\theta = 10^\circ \Rightarrow \beta \approx 27.5^\circ$

(From chart on inside cover of text)

2) $M_{2A} = f(M_{1A})$ $M_{1A} = M_1 \sin \beta = 3 \sin(27.5^\circ) = 1.385$

$\Rightarrow M_{2A} = 0.7483$ (From table A.2 using $M = 1.385$)

3) $\Rightarrow M_2 = \frac{M_{2A}}{\sin(\theta - \theta)} = \frac{0.7483}{\sin(17.5^\circ)} = 2.49$

4) $\Rightarrow \frac{P_{02}}{P_{0A}} = f_n(M_{2A}) = 0.52$ (From TBL. A.2 using $M_1 = 2.49$)

5) $\frac{P_{0A}}{P_{01}} = f_n(M_{1A}) \approx 0.963$ (" " " " " " = 1.385)

$\Rightarrow P_{0A} = P_{01} (0.963) = 96.3 \text{ kPa}$

6) $P_{02} = (0.52)(P_{0A}) = 82.05 \text{ kPa}$